

# DELAMINATION OF COMPOSITE STRUCTURES AND FAILURE MODES OF BONDED ELEMENTS

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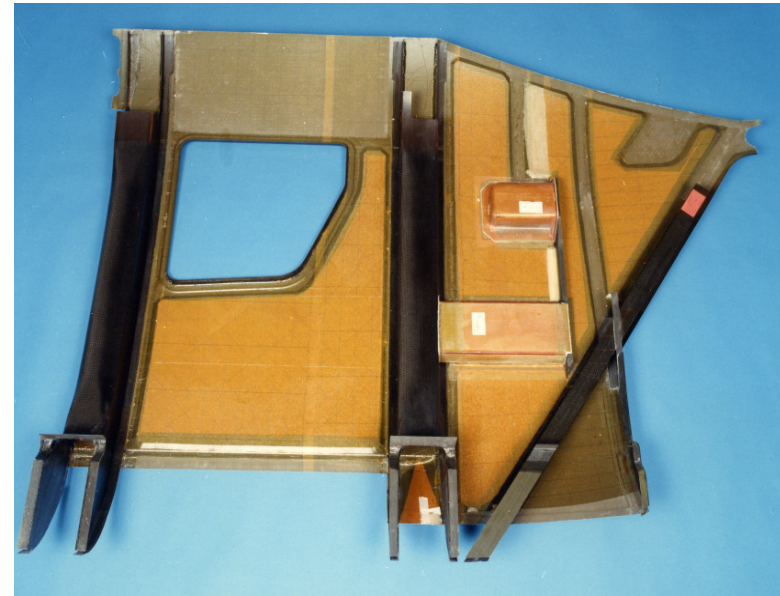
# Table of Contents

- **Introduction**
- **Examples for Bonded Helicopter Structures**
- **Stress Analysis According to the “Transfer-Matrix” Method**
- **The “Three-Beam-Structure” with Different Loading Situations**
- **Tension and Torsion Loaded Tapered and Bonded Laminates**

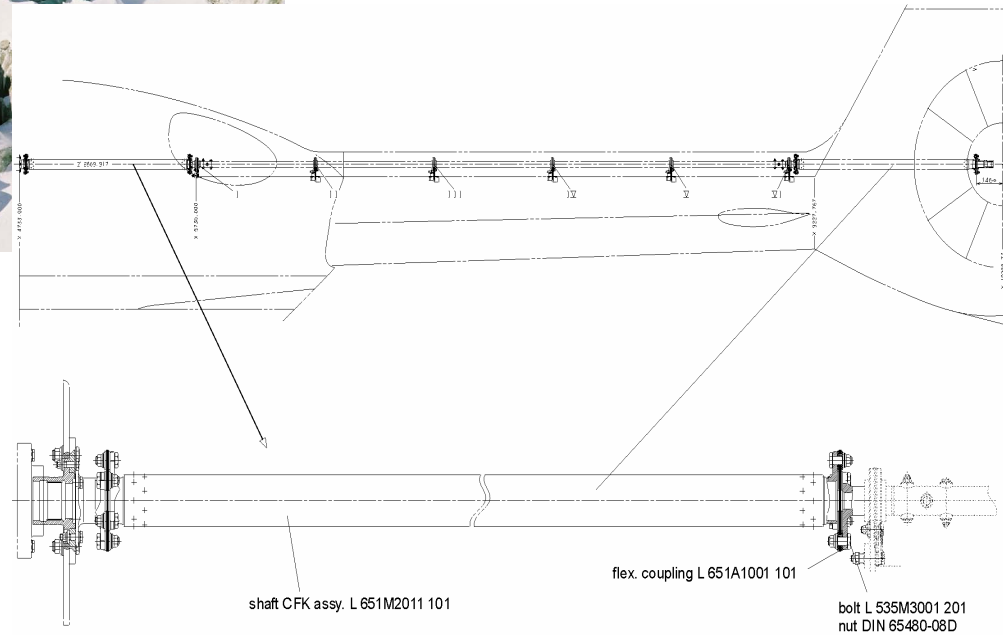
# The Commercial Helicopter BK 117 with the Rotor Blade Attachment Fitting and Metallic Erosion Protection



# The BK 117 with Composite Fuselage and Side Shell with Bonded Elements



# The EC 135 with „Fenestron“ Drive Shafts



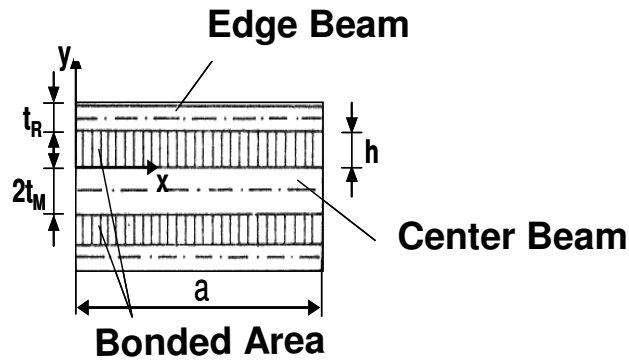
# The EC 135 in Life Saving Action with the Impacted Drive Shaft after Static Residual Strength Test



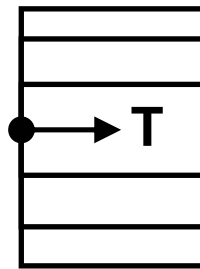
The EC 135 helicopter and the “Fenestron” carbon composite drive shaft

# **Stress Analysis According to the “Transfer-Matrix” Method**

# The „Three-Beam-Structure“, the Stress Situation in the Bonded Area, the Transfer Directions and the Solution of the Differential Equation

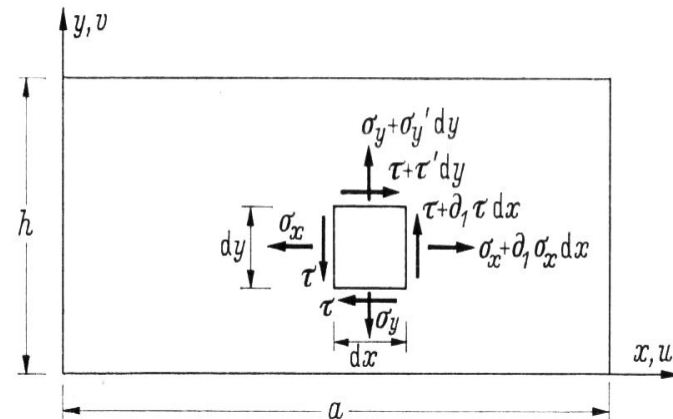


Edge 0                      Edge 1

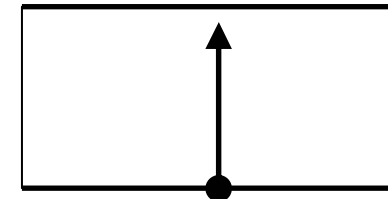


Differential-Equation-System:

$$\partial_1 w = A w ,$$



**T**



Edge b

Edge a

Transfer-Matrix **T**:

$$T = e^A = E + A + A^2/2! + A^3/3! + \dots$$



# The Relation between the Stresses, Deformations and Strains in the Bonded Area

a) The Conditions of Equilibrium:

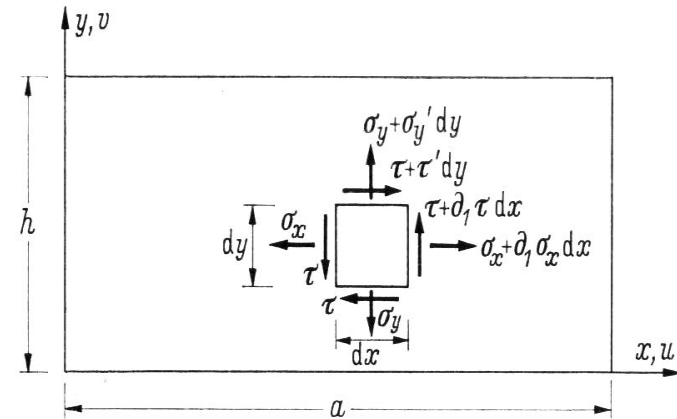
$$\partial_1 \sigma_x + \tau' = 0 \quad \sigma_y' + \partial_1 \tau = 0$$

$\sigma_x, \sigma_y$  are the Normal Stresses,  $\tau$  is the Shear Stress

b) The Conditions of Deformations

$$\varepsilon_x = \partial_1 u, \quad \varepsilon_y = v', \quad \gamma = u' + \partial_1 v$$

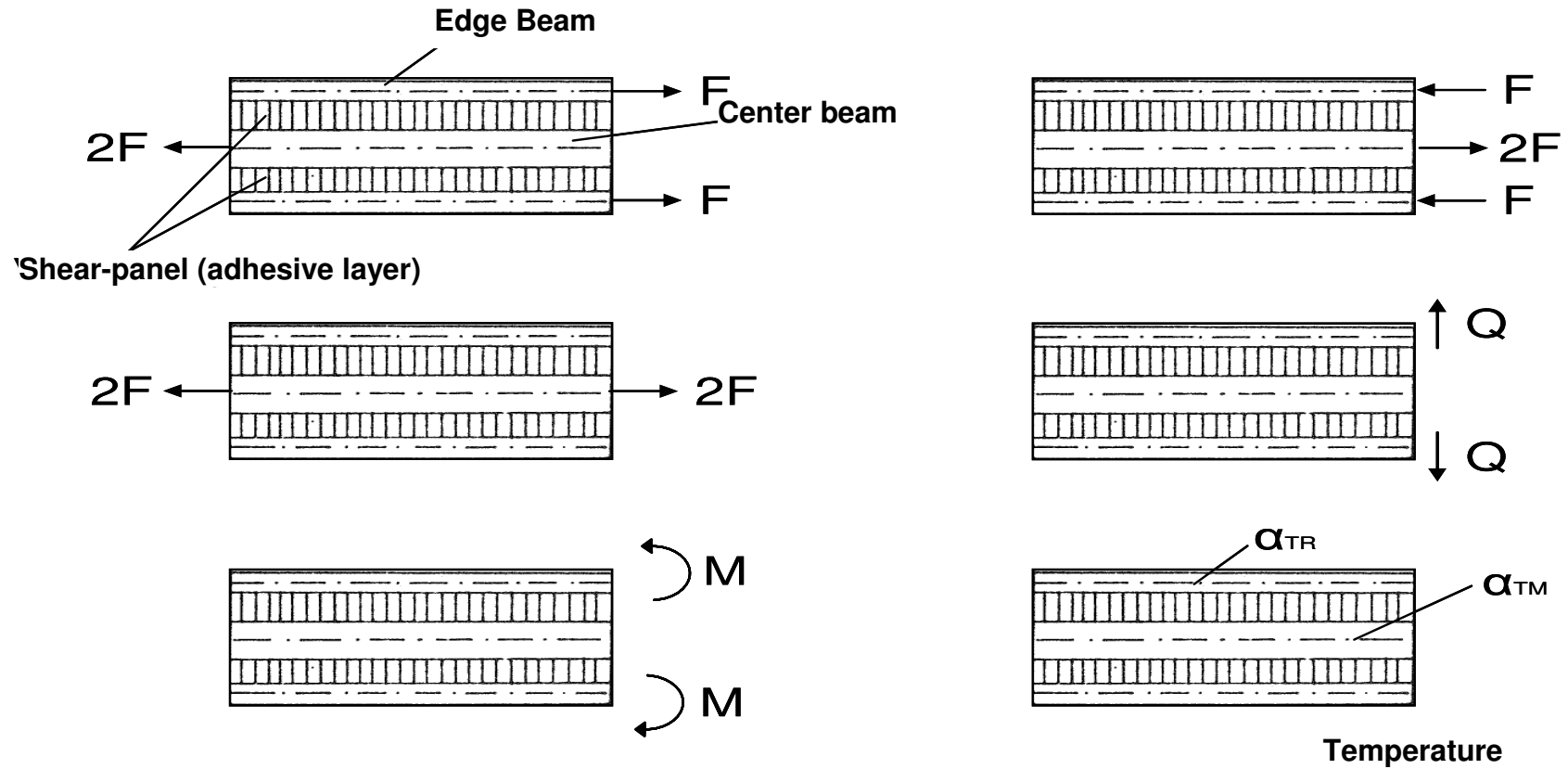
$u, v$  are the Deformations,  $\varepsilon_x, \varepsilon_y$  are the Strains and  $\gamma$  the Shear Angle



c) The Relation between a) and b) can be established with the help of the Elasticity-Law with  $\nu = 0, E_x = 0$  :

$$\sigma_x = 0, \quad \sigma_y = E_y \varepsilon_y, \quad \tau = G\gamma \begin{pmatrix} u \\ v \\ \sigma_y \\ \tau \end{pmatrix}' = \begin{pmatrix} 0 & -\partial_1 & 0 & 1/G \\ 0 & 0 & 1/E_y & 0 \\ 0 & 0 & 0 & -\partial_1 \\ 0 & 0 & 0 & 0 \end{pmatrix} \begin{pmatrix} u \\ v \\ \sigma_y \\ \tau \end{pmatrix} \quad (1)$$

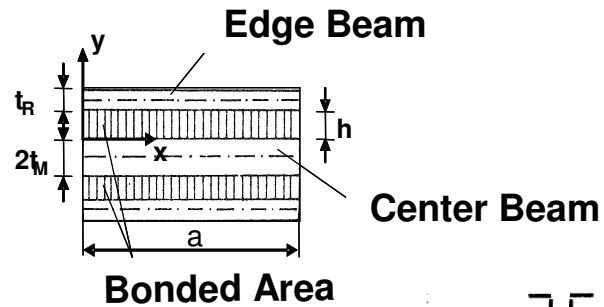
# Different Basic Loading Conditions



The “Three-Beam-Structure” with different loading situations

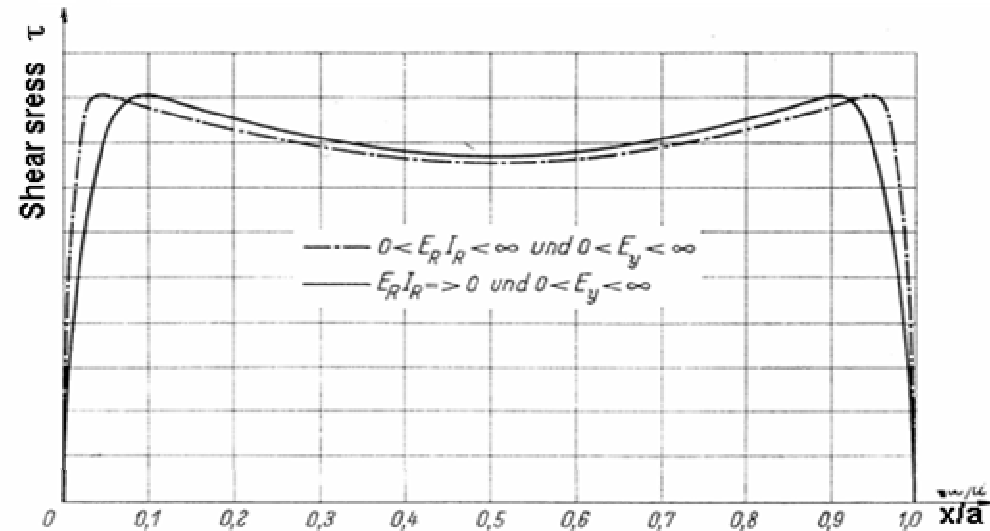
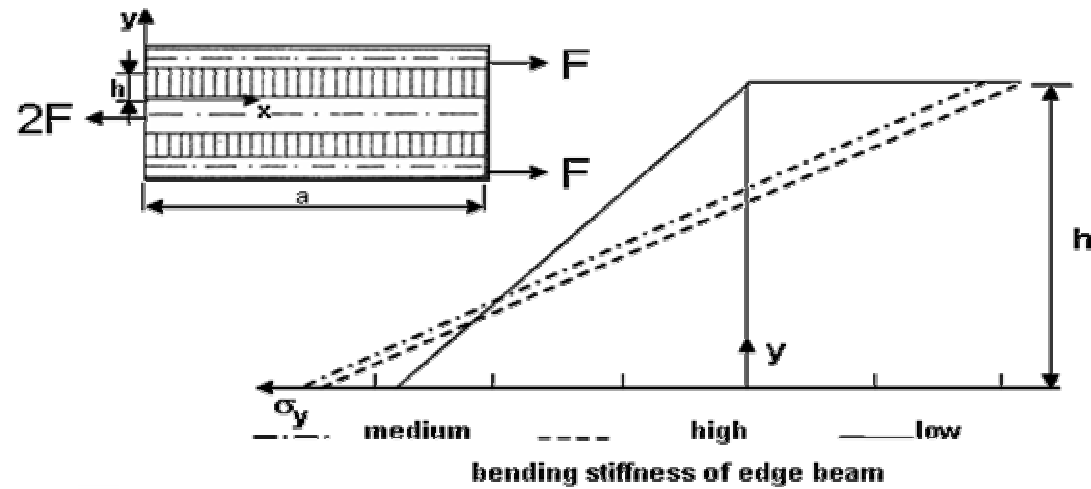
# The Differential-Equation-System of the 1. Order for the „Three-Beam-Structure“

with  $k = -6a(t_R/2 + h/3)/h^2$



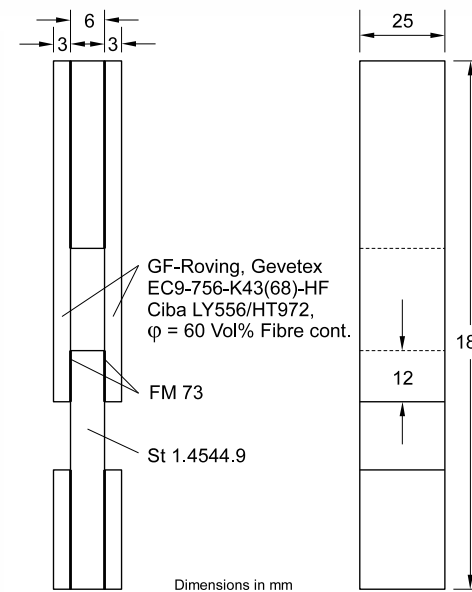
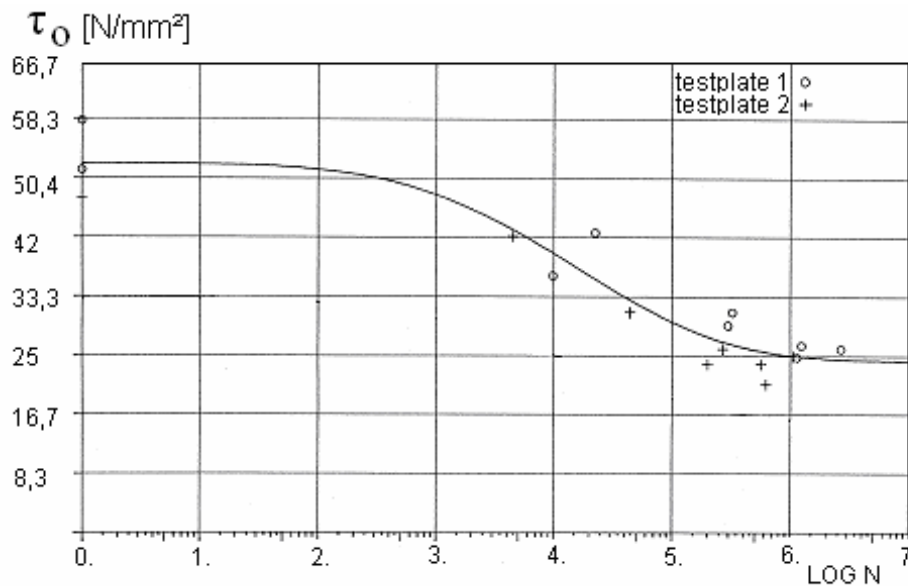
$$\bar{\partial}_1 \begin{bmatrix} \bar{v}_R \\ \bar{\psi}_R \\ \bar{M}_R \\ \bar{Q}_R \\ \bar{u}_R \\ \bar{N}_{x,R} \\ \bar{\tau}_0 \\ \bar{\sigma}_{y_0} \\ \bar{u}_M \\ \bar{N}_{x,M} \end{bmatrix} = \begin{bmatrix} 1 & & & & & & & & & \\ & 1 & & & & & & & & \\ & & 1 & & & & & & & \\ & \frac{2 a^4 E_y}{h E_R I_R} & & & & & & & & \\ & & & & 1 & & & & & \\ & & & & & \frac{a G}{t_R E_R} & & & & \\ \frac{2 a^2 E_y}{h^2 G} & & & & & & \frac{2 a E_y}{h G} & & & \\ & k & & -6 \frac{a^2}{h^2} & & 6 \frac{a}{h} & & 6 \frac{a^2}{h^2} & & \\ & & & & & & & & 1 & \\ & & & & & \frac{a G}{E_M t_M} & & & & \end{bmatrix} \begin{bmatrix} \bar{v}_R \\ \bar{\psi}_R \\ \bar{M}_R \\ \bar{Q}_R \\ \bar{u}_R \\ \bar{N}_{x,R} \\ \bar{\tau}_0 \\ \bar{\sigma}_{y_0} \\ \bar{u}_M \\ \bar{N}_{x,M} \end{bmatrix}$$

# The Shear Stress Distribution for a Double Lap Joint Including the Transverse Stresses for Different Bending Stiffnesses of the Upper Beam

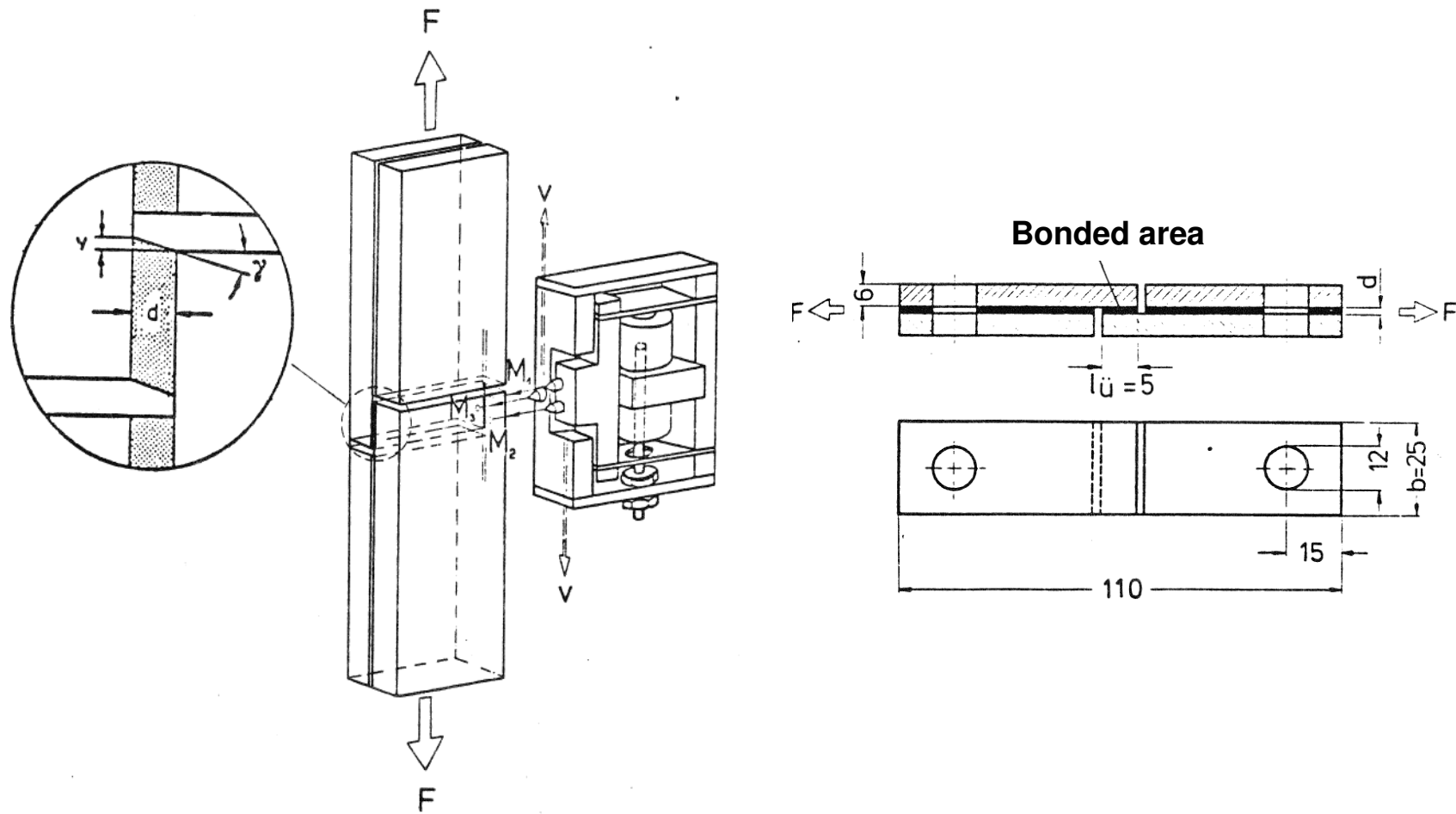


# Stress Results of Double-Lap-Joints Loaded by Dynamic Tension for Different Material Combinations and the Adhesive FM 73

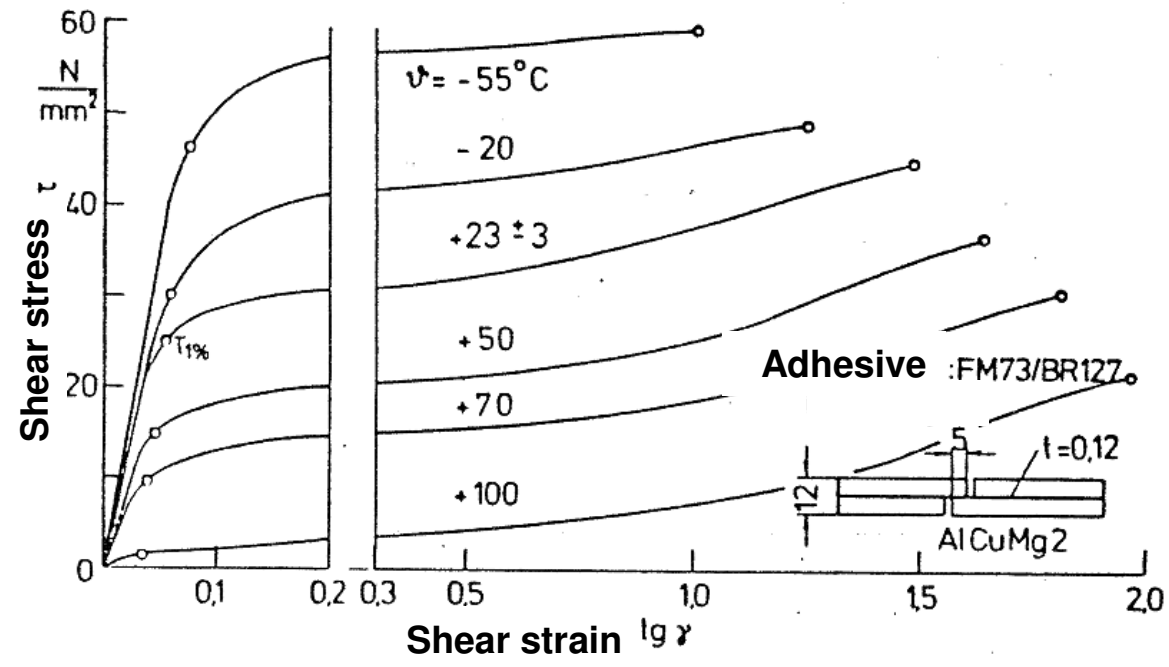
Material Combination	thickness of adhesive layer [mm]	factor for max. shear stress j [-]	max. static shear strength [N/mm <sup>2</sup> ]	G <sub>ci</sub> (static) [N/mm]	max. dyn. shear strength (10 <sup>7</sup> cycles) [N/mm <sup>2</sup> ]	G <sub>ci</sub> (dynamic) [N/mm]
Ti/GFC	0,2	1,7	52,9	0,40	24,8	0,09
St/GFC	0,13	2,3	67,3	0,42	37,9	0,14
Al/Al	0,19	1,4	48,3	0,33	25,7	0,09



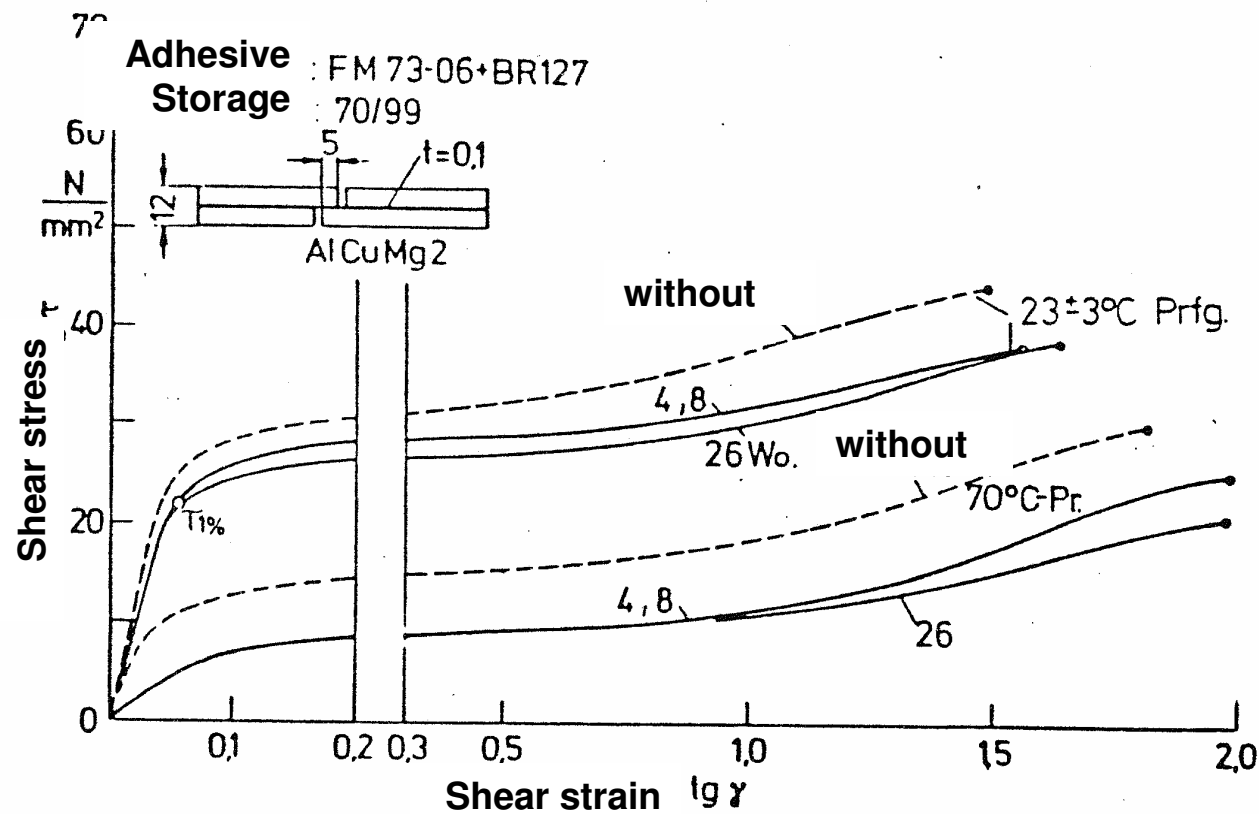
# Test Equipment and Set-up for the Determination of the Shear-Strength and Shear-Strain Curves with the Help of Single-Lap Specimens



# Shear-Strength and Shear-Strain Curves at different Temperatures (Adhesive: FM 73)



# Shear-Strength and Shear-Strain Curves with and without Longtime Storage at hot/wet Conditions 70/99 (Kleber: FM 73)



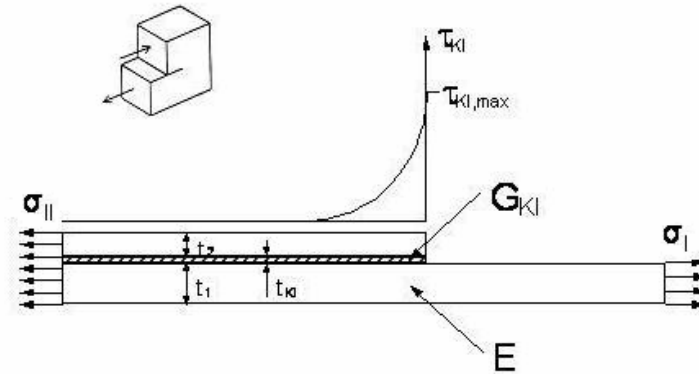


# Tension and Torsion Loaded Tapered and Bonded Laminates



The Commercial Helicopter EC 135 with the „Flex-Beam“ Attachment Area of the Blade

# Bonded Erosion Strip, analyzed with the help of the „Shear-Lag“ Theory and the „Strain Energy Release Rate“



$$\tau_{KI,max} = \sigma_{II} \cdot \sqrt{\frac{G_{KI}}{t_{KI}}} \cdot \sqrt{\frac{t_2 \cdot (t_1 + t_2)}{E \cdot t_1}}$$

$$\sigma_{II} = \tau_{KI,max} \sqrt{\frac{t_{KI}}{G_{KI}}} \cdot \sqrt{\frac{E \cdot t_1}{t_2 \cdot (t_1 + t_2)}}$$

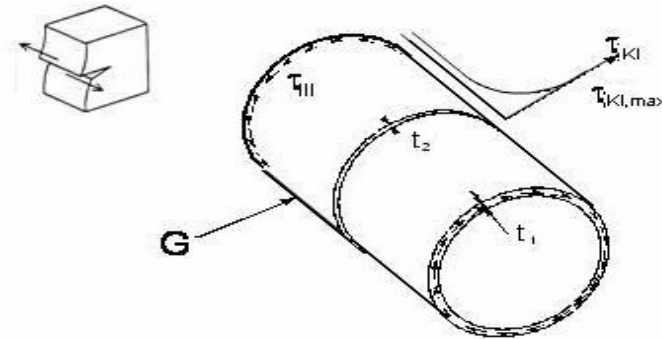
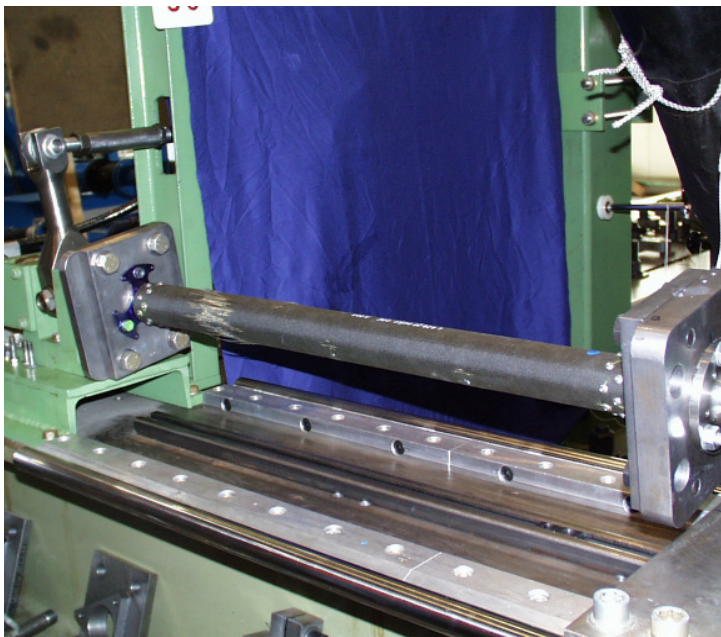
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$$\sigma_{II} = \sqrt{2 \cdot G_{C,II}} \cdot \sqrt{\frac{E \cdot t_1}{t_2 \cdot (t_1 + t_2)}}$$


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$$G_{C,II} = \frac{1}{2} \cdot \tau_{KI,max}^2 \cdot \frac{t_{KI}}{G_{KI}}$$

# Bonded and Riveted Drive Shaft, analyzed with the help of the „Shear-Lag“ Theorie and the „Strain Energy Release Rate“



$$\tau_{KI,max} = \tau_{III} \cdot \sqrt{\frac{G_{KI}}{t_{KI}}} \cdot \sqrt{\frac{t_2 \cdot (t_1 + t_2)}{G \cdot t_1}}$$

$$\tau_{III} = \tau_{KI,max} \sqrt{\frac{t_{KI}}{G_{KI}}} \cdot \sqrt{\frac{G \cdot t_1}{t_2 \cdot (t_1 + t_2)}}$$

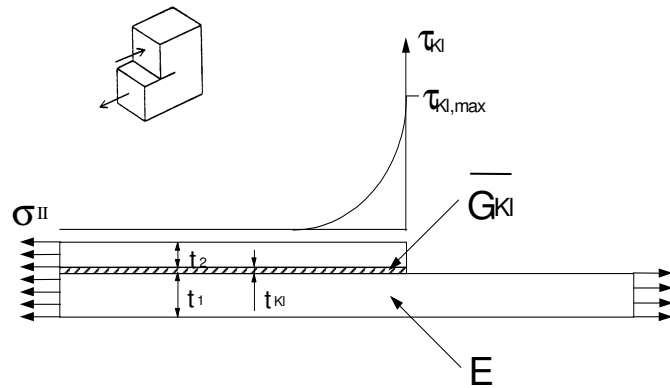
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$$\tau_{III} = \sqrt{2 \cdot G_{C,III}} \cdot \sqrt{\frac{G \cdot t_1}{t_2 \cdot (t_1 + t_2)}}$$


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$$G_{C,III} = \frac{1}{2} \cdot \tau_{KI,max}^2 \cdot \frac{t_{KI}}{G_{KI}}$$

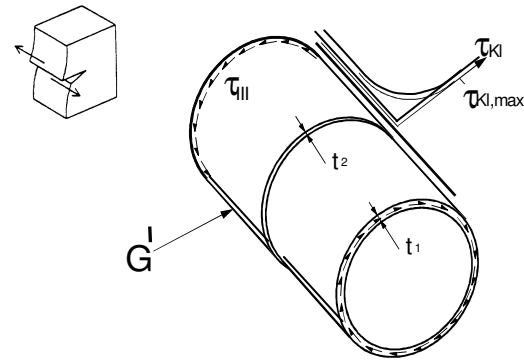
# Tapered/Bonded Structure Loaded in Tension and in Torsion



$$\sigma_{II} = \tau_{Kl,max} \cdot \sqrt{\frac{t_{Kl}}{G_{Kl}}} \cdot \sqrt{\frac{E \cdot t_1}{t_2 \cdot (t_1 + t_2)}}$$

$$\sigma_{II} = \sqrt{2 \cdot G_{C,II}} \cdot \sqrt{\frac{E \cdot t_1}{t_2 \cdot (t_1 + t_2)}}$$

$$G_{C,II} = \frac{1}{2} \cdot (\tau_{Kl,max})^2 \cdot \frac{t_{Kl}}{G_{Kl}}$$



$$\tau_{III} = \tau_{Kl,max} \cdot \sqrt{\frac{t_{Kl}}{G_{Kl}}} \cdot \sqrt{\frac{G' \cdot t_1}{t_2 \cdot (t_1 + t_2)}}$$

$$\tau_{III} = \sqrt{2 \cdot G_{C,III}} \cdot \sqrt{\frac{G' \cdot t_1}{t_2 \cdot (t_1 + t_2)}}$$

$$G_{C,III} = \frac{1}{2} \cdot (\tau_{Kl,max})^2 \cdot \frac{t_{Kl}}{G_{Kl}}$$

# Summary

- **Bonded structures are widely used for highly loaded helicopter elements**
- **Examples for bonded helicopter elements are e.g. the metallic erosion strips bonded to the composite rotor blades**
- **Stress analysis according to the “Transfer-Matrix” method is a important analytical tool for calculating shear stresses**
- **Tension and torsion loaded tapered and bonded laminates can also be analyzed by energy methods (Strain-Energy-Release-Rate)**
- **Stress-strain curves are important for the calculation of shear stresses in an inelastic range**

**Acknowledgment:** The Author would like to thank Eurocopter Germany for providing the fotos.

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